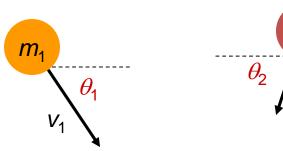
# ESTIMATING AV

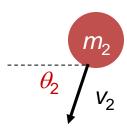
Sudipto Mukherjee





### **Conservation of Momentum in 2-D**

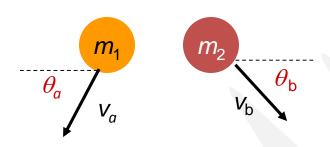




#### before:

$$\mathbf{p}_{\mathbf{X}} = m_1 v_1 \cos \theta_1 - m_2 v_2 \cos \theta_2$$

$$\mathbf{p}_{V} = m_1 v_1 \sin \theta_1 + m_2 v_2 \sin \theta_2$$



#### after:

$$\mathbf{p}_{X} = -m_1 v_a \cos \theta_a + m_2 v_b \cos \theta_b$$

$$p_y = m_1 v_a \sin \theta_a + m_2 v_b \sin \theta_b$$

Conservation of momentum equations:

$$m_1 v_1 \cos \theta_1 - m_2 v_2 \cos \theta_2 = -m_1 v_a \cos \theta_a + m_2 v_b \cos \theta_b$$
  

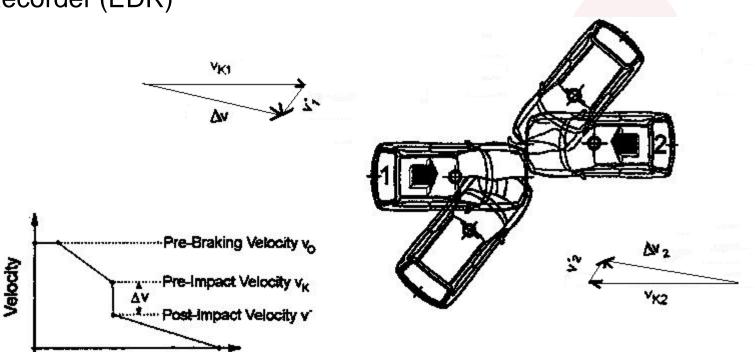
$$m_1 v_1 \sin \theta_1 + m_2 v_2 \sin \theta_2 = m_1 v_a \sin \theta_a + m_2 v_b \sin \theta_b$$



Time

## What is DeltaV ( $\Delta V$ )

- Delta-V is the change of velocity of the center of gravity of a vehicle in the road fixed coordinate system during the contact phase
- Different from the change of velocity calculated by the integration of the vehicle deceleration components measured in the Event Data Recorder (EDR)





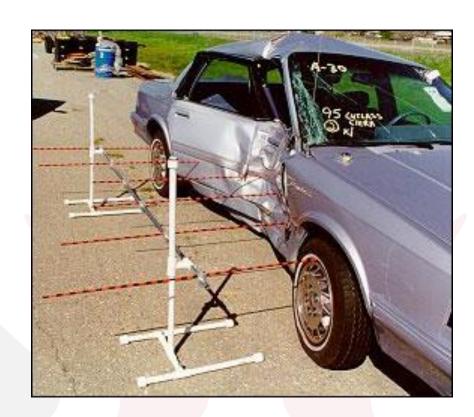
#### **CRASH3 and WinSMASH**

- The National Highway and Transport Safety Association (NHTSA) evolved CRASH3 in the 1980's and more recently WinSMASH
- Trajectory Analysis:
  - + Analyze the ΔV using detailed scene measurement and conservation of momentum
- Damage analysis
  - + ΔV has a linear relationship with residual damage
- Satisfactory agreement is supposed to exist between the two estimates when their delta-V components differ by no more than 4 kmph or ten percent, whichever is greater.
- In user friendly form available as AiDamage™
- Output is
  - + Three components-x, y and angular: ΔV
  - + Energy dissipated (mainly) in crushing the vehicular structure



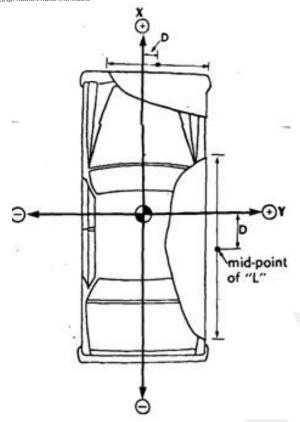
## **Crush Measurement**

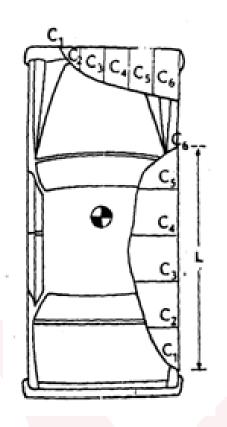
- A measure of the energy absorbed and hence its effectiveness in fulfilling its design objective
- An estimate of the change in the speed of both vehicles (or one vehicle for say a tree or a barrier)
- Estimating the energy and delta-Vs
- Need a rig with lasers or use old fashioned tape and plumbline.





#### Make a sketch

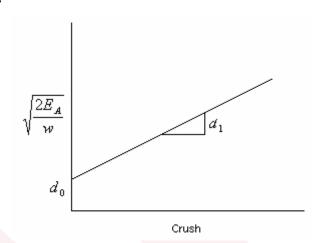




This data is often reported in the Collision Deformation
 Classification (CDC) as per SAE J224 MAR80

## **Unitized body cars**

- Linear relationship between the impact velocity and crush was observed from the crash tests conducted on 1971-1974 General Motors full frame body cars.
- It has been experimentally established that residual crush is linearly proportional to  $\sqrt{(2E_A/w)}$  for unitized body cars
- E<sub>A</sub> is the energy absorbed by the vehicle structure and w is the width of the crush.



Over a crush profile 
$$c(w)$$
: 
$$E_w = \int_0^w \frac{1}{2} (d_0 + d_1 \times C)^2 dw$$



#### **Vehicle Data Needed**

Туре	Wheelbase
1	<= - 240.8
2	240.8 – 258.0
3	258.0 – 280.4
4	280.4 – 298.4
5	298.4 – 312.9
6	> - 312.9
7(vans)	276.8 – 330.2

k = 0.3 x (vehicle length)

The generic d<sup>0</sup> and d<sup>1</sup> stiffness coefficients may not apply for bumper over-ride and under-ride crashes for example

	Front		Rear		Side	
Cat.	$\mathbf{d}_0$	$\mathbf{d_1}$	$\mathbf{d}_0$	$\mathbf{d_1}$	$\mathbf{d}_0$	$\mathbf{d_1}$
	√Newton	√Newton	$\sqrt{\text{Newton}}$	$\sqrt{\text{Newton}}$	$\sqrt{\text{Newton}}$	√Newton
		cm		cm		cm
1	91.4	6.7	93.88	5.43	63.3	6.83
2	97.0	7.22	96.23	5.28	63.3	8.02
3	102.1	7.25	99.49	5.56	63.3	7.50
4	107.0	6.36	99.99	5.37	63.3	7.21
5	109.6	6.18	99.97	4.50	63.3	5.19
6	116.0	5.75	74.86	6.94	63.3	5.69
7	109.7	8.51	98.69	7.79	-	_
(vans)	20011		7 0107			
8 (pickup)	105.7	7.98	101.42	7.77	-	-
9 (FWD)	99.18	6.46	-	-	-	-

# **Computing Delta V**

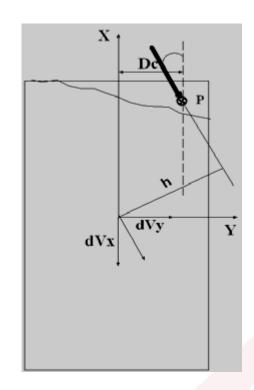
- The total energy,  $E_T$  = sum of energy absorbed by each vehicle.
- The radius of gyration of vehicles 1 and 2 are k<sub>1</sub> and k2 and the moment arm of impact force h1 and  $h_2$ .

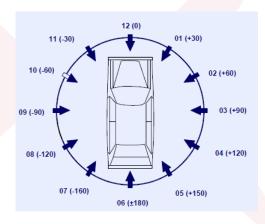
$$\Delta V_{1} = \sqrt{\frac{2E_{T}\gamma_{1}}{M_{1}\left(1 + \frac{\gamma_{1}M_{1}}{\gamma_{2}M_{2}}\right)}} \qquad \gamma_{1} = \frac{k_{1}^{2}}{k_{1}^{2} + h_{1}^{2}}$$

$$\gamma_{2} = \frac{k_{2}^{2}}{k_{2}^{2} + h_{2}^{2}}$$

$$\Delta V_2 = \sqrt{\frac{2E_T \gamma_2}{M_2 \left(1 + \frac{\gamma_2 M_2}{\gamma_1 M_1}\right)}}$$

$$\gamma_1 = \frac{k_1^2}{k_1^2 + h_1^2}$$
$$\gamma_2 = \frac{k_2^2}{k_2^2 + h_2^2}$$







## Help for you: modern packages

- Impact velocity
- Position of impact (POI) in x-y direction
- Contact plane angle phi (degrees CCW from the global x-axis)
- 4. Pre-impact directions
- 5. Vehicle positions
- 6. Coefficient of Restitution
- 7. Contact plane friction

Minimise the weighted error of these parameters.

$$Q = \sqrt{\frac{\sum_{i} (W_{i}.X_{i})^{2}}{\sum_{i} W_{i}^{2}}}.100\%$$

Using Gauss-Sidel, GA or Monte Carlo in isolation or in combination

## **Summary**

$$\Delta V_1 = \sqrt{\frac{2E_T \gamma_1}{M_1 \left(1 + \frac{\gamma_1 M_1}{\gamma_2 M_2}\right)}}$$

$$\Delta V_2 = \sqrt{\frac{2E_T \gamma_2}{M_2 \left(1 + \frac{\gamma_2 M_2}{\gamma_2 M_2}\right)}}$$

- Measure the crush profiles and centre of deformation.
- Estimate direction of approach to estimate h.
- Lookup values of m and k for both cars.
- Estimate  $d_0$  and  $d_1$  from the chart or measured data.
- Ready to find  $\Delta V$  manually or using a package.
- Do remember this is a gross estimate and has to be verified and tuned through trajectory reconstruction.